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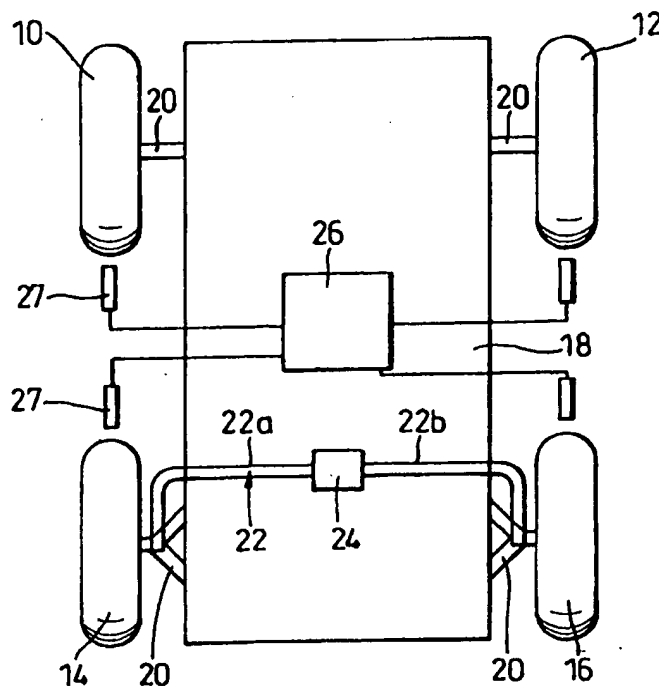
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(21) International Application Number: PCT/GB99/01714 (22) International Filing Date: 28 May 1999 (28.05.99) (30) Priority Data: 9812264.1 9 June 1998 (09.06.98) GB (71) Applicant (for all designated States except US): ROVER GROUP LIMITED [GB/GB]; International Headquarters, Warwick Technology Park, Warwick CV34 6RG (GB). (72) Inventors; and (75) Inventors/Applicants (for US only): BURDOCK, William [GB/GB]; 106 East View Road, Sutton Coldfield, West Midlands B72 1JA (GB). CLARE, David, Andrew [GB/GB]; 5 Birch Grove, Wellesbourne, Warwickshire CV35 9SJ (GB). (74) Agents: BURRAGE, David, John et al.; Rover Group Limited, Gaydon Test Centre, Patent Dept., Banbury Road, Lighthorne, Warwick CV35 0RG (GB).		(81) Designated States: AU, GB, US, ZA, European patent (AT, BE, CH, CY, DE, DK, ES, FI, FR, GB, GR, IE, IT, LU, MC, NL, PT, SE). Published With international search report.	

(54) Title: VEHICLE ROLL CONTROL

(57) Abstract

An active vehicle roll control system is disclosed in which a roll bar (22) has two halves (22a, 22b) which can be locked together to allow the system to operate passively. A sensing system based on a lateral accelerometer and vehicle speed sensor is arranged to detect when the vehicle is on a side slope and to put the roll control system into the passive mode.



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### Vehicle Roll Control

The present invention relates to active suspension systems for vehicles, and in particular to such systems which include active roll control.

It can be a problem with active suspension control systems, particularly in off-road vehicles that they do not respond suitably when the vehicle is traversing a steep side slope. This is because, in conventional systems, the presence of a side slope cannot be detected and the system therefore responds to a side slope as if the vehicle were cornering. This can result in the consumption of a lot of power as the system tries to compensate for what can be the equivalent of severe vehicle roll but which can last for long periods of time. This is because the vehicle can continue to traverse the side slope for a considerable length of time whereas severe roll is generally only present for short periods.

It is known from EP 0 283 004 to provide a vehicle roll control system in which the roll control mechanism is modified in dependence on the speed of the vehicle. However, this system still does not address the problem of roll control on a side slope.

The present invention provides apparatus for detecting traversal of a side slope by a vehicle, the apparatus including an accelerometer arranged to produce an acceleration signal indicative of the lateral acceleration of the

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vehicle, speed measuring means arranged to produce a speed dependent signal dependent on the speed of the vehicle and control means arranged to monitor the acceleration signal and the speed dependent signal and to produce a side slope detection signal based on a comparison between them.

- 5       The speed dependent signal can vary only with the vehicle speed. Alternatively, if the vehicle includes a sensor for measuring steering angle the speed dependent signal can also be dependent on the steering angle.

Preferably the speed dependent signal is arranged to be vary with to the square of the vehicle speed. This gives the best indication of cornering  
10   acceleration because the cornering acceleration varies with the square of the speed of the vehicle.

Preferably the apparatus further comprises comparator means arranged to produce a difference signal indicative of the difference between the speed dependent signal and the acceleration signal.

- 15       Preferably the apparatus further comprises integrating means arranged to integrate the difference signal over time and to produce the side slope detection signal if the integral reaches a predetermined value.

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The present invention further provides vehicle suspension system including roll control means for controlling roll of the vehicle, the roll control means having active and passive mode, the system further comprising side slope detection means arranged to detect when the vehicle  
5 is traversing a side slope, wherein the roll control means is arranged to respond to detection of a side slope by operating in the passive mode.

Preferably the roll control means includes a hydraulic circuit including a pump which is arranged to provide hydraulic pressure for use in active roll control when the roll control means is in the active state but which can be  
10 turned off or operate at reduced power when the roll control means is in the passive state.

The roll control means may include a roll bar having two parts which can be moved relative to each other to control roll when the roll control means is in the active state and which can be locked together when the roll  
15 control system is in the passive state.

Preferably the side slope detection means comprises side slope detection apparatus according to the invention.

Preferably the roll control means is arranged to switch from the passive mode to the active mode if the vehicle speed exceeds a predetermined

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threshold speed while the roll control means is in the passive mode. This is because at higher speeds it can be assumed that the vehicle is no longer on a steep side slope and that normal roll control will be required.

Preferably the system further comprises means for producing a high  
5 speed signal when the vehicle speed exceeds said threshold speed, and wherein the high speed signal is input to the comparator means which is arranged to respond by altering the difference signal as if a speed signal indicative of a high vehicle speed or a lateral acceleration signal indicative of low lateral acceleration had been received.

10 Preferred embodiments of the present invention will now be described by way of example only with reference to the accompanying drawings in which:

Figure 1 is a diagrammatic representation of a vehicle including a suspension according to an embodiment of the invention,

15 Figure 2 is a plan view of the vehicle of Figure 1 whilst cornering,

Figure 3 is an end view of the vehicle of Figure 1 whilst traversing a side slope,

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Figure 4 is a diagrammatic representation of the control unit of the system of Figure 1, and

Figure 5 is a graph showing changes in time in the output from part of a control circuit of the vehicle of Figure 1.

5 Referring to Figure 1, a vehicle has four wheels 10, 12, 14, 16 each mounted on the vehicle body 18. The vehicle has an independent suspension, each of the wheels being attached to the body 18 through a suspension arm 20 so that it can move vertically relative to the body 18. A roll bar 22 is connected between the two rear wheels 14, 16 to control the  
10 roll of the rear of the vehicle. The roll bar 22 is split in the middle into two halves 22a, 22b which can be rotated relative to each other by a rotary actuator 24 under the control of a control unit 26. This enables vehicle roll to be controlled actively in response to signals input to the control unit from wheel speed sensors 27 and a number of accelerometers which provide  
15 signals indicative of the acceleration of parts of the vehicle body in various directions. A similar roll bar, which is not shown, would also normally be connected between the front wheels 10, 12.

Referring to Figure 2, one of the sensors is a lateral accelerometer 30 arranged to measure the acceleration of the vehicle body in the lateral  
20 direction, i.e. the horizontal direction perpendicular to the normal direction

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of travel. As shown in Figure 2, when the vehicle is cornering the lateral accelerometer 30 detects a lateral acceleration  $a$  which is given by the formula  $a = v^2 / r$  where  $a$  is the acceleration,  $v$  is the speed of the vehicle in the forward direction and  $r$  is the radius of the arc through which the  
5 vehicle is turning.

Referring to Figure 3, when the vehicle is travelling on a side slope at an angle  $\theta$  to the horizontal the accelerometer measures a lateral acceleration  $a$  which is given by the formula  $a = g \sin\theta$  where  $g$  is the acceleration due to gravity ( $9.8 \text{ ms}^{-2}$ ).

10 The system is required to distinguish between when the vehicle is traversing a steep side slope, which will generally only happen at low vehicle speeds when the vehicle is travelling off road, and when it is cornering hard, which will generally only happen at higher speeds on road.

Referring to Figure 4, the control unit 26 has one input 32 which  
15 receives a lateral acceleration signal from the lateral accelerometer 30 and another set of inputs 34 which receive signals from the wheel speed sensors 27. The lateral acceleration signal is input to a comparator 36 and compared with a threshold signal. If the measured lateral acceleration is greater than the threshold value  $g_t$ , the comparator 36 closes a switch 38 connecting the



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lateral acceleration signal to a first input 40 of a comparator 42 via an amplifier 44 having a gain G1.

The four wheel speed signals are input to a circuit 46 which calculates the vehicle speed from them in a known manner and produces a vehicle speed signal proportional to the speed of the vehicle. The vehicle speed  
5 speed signal is input to a squaring device 47 which produces a squared signal which varies with the square of the vehicle speed, and which is input via an amplifier 48 of gain G2 to a second input 50 of the comparator 42.

The road speed signal is also input to a comparator 52 which compares  
10 it with a threshold speed signal. If the vehicle speed is above the threshold speed determined by the threshold signal, the comparator 52 produces a high speed signal which is input to a third input 54 of the comparator 42 via an amplifier 56 of gain G3.

Assuming that the vehicle speed is below the threshold speed such that  
15 there is no input to the third input 54 of the comparator, the comparator 42 compares the signals at its first and second inputs 40, 50 and produces a difference signal equal to the difference between the acceleration signal and the squared speed signal. This difference signal therefore gives an indication of the instantaneous side slope inclination being traversed by the  
20 vehicle. Clearly to provide an exact instantaneous measurement of side

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slope the steering angle of the front steerable wheels of the vehicle would need to be taken into account. However this requires a separate steering angle sensor, which adds to the cost of the system, and the arrangement described above has been found to be sufficiently accurate for most  
5 purposes.

The difference signal, which is positive when the lateral acceleration signal is greater than the squared speed signal and negative when the squared speed signal is greater than the lateral acceleration signal, is then input to an integrator 58 which continually integrates the difference signal  
10 over time. The integrator is arranged so that the integral signal can only have positive values. When it falls to zero it remains there until the difference signal is positive again. The integral signal is also limited to a maximum value so that, when the vehicle comes off a side slope, the integral signal will fall to zero again within a reasonable period.

15 If the vehicle speed is greater than the threshold speed the high speed signal is input to the comparator 42 via the amplifier 56. The gain G3 of this amplifier is relatively high compared to that G1, G2 of the other two amplifiers. The comparator deducts the value of the high speed signal from the difference between the lateral acceleration signal and the squared speed  
20 signal to produce a modified difference signal. The result of this is that, if the vehicle speed exceeds the threshold speed the difference signal goes to a

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relatively high negative value causing the integral signal to drop rapidly to zero.

The lateral acceleration is also monitored by a comparator 60 and if it exceeds a predetermined high level  $g_m$  a signal to the integrator 58 resets the system to normal operation. This level is set to be slightly less than the maximum which can be experienced on a side slope without the vehicle rolling over. If this level of lateral acceleration is experienced it must either be due to some manoeuvre other than the traversing of a side slope, in which case normal roll control will be required, or to an extreme side slope, in which case the normal anti-roll control may help to prevent the vehicle from rolling over.

It will be appreciated that the integral signal will vary over time dependent on the cornering and side slope which the vehicle experiences in a manner illustrated in Figure 5.

Referring to Figure 5, which shows variations in the integral signal  $V$  over time, between times  $t_0$  and  $t_1$  the vehicle is travelling over a rough off-road surface with little permanent side slope. Under these conditions when there is a high measured lateral acceleration it is due to cornering or unevenness in the terrain and does not last for long periods. Therefore when the difference signal does go positive, it quickly returns to zero. At time  $t_1$

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the vehicle encounters a side slope so the measured lateral acceleration rises and remains at a high level whilst still varying due to vehicle roll and cornering. The difference signal therefore starts to rise. When the difference signal reaches a threshold value  $V_t$  the control unit 26 determines that the vehicle is on a side slope and locks the two halves of the rotary actuator 24 together. This locks the two halves of the anti-roll bar together so that it acts like a normal, one-piece anti-roll bar. This puts the roll control system into passive mode and effectively reduces the power required by the rotary actuator to zero. At time  $t_2$  the integral signal reaches a maximum value  $V_m$  and is held there while the difference signal remains positive. At time  $t_3$  the vehicle comes off the side slope and the lateral acceleration therefore drops. The difference signal becomes negative and the integral signal starts to fall. At time  $t_4$  the vehicle speed exceeds the threshold speed so the high speed signal is input to the comparator 42 and the difference signal takes a high negative value. The integral signal therefore drops rapidly to zero. As the integral signal passes through the threshold value  $V_t$  the rotary actuator 24 is unlocked and the roll control system becomes active again.

It will be appreciated that the control algorithms described above can be used in various types of active roll control systems. For example the system described in our co-pending international application No. PCT/GB97/03314 which uses a hydraulic linear actuator in place of the rotary actuator described above can include the features of the present

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invention. It can also be applied in fully hydraulic or pneumatic suspension systems in which the active roll control is provided by applying controlled fluid pressure to vertical actuators at each wheel.

CLAIMS

1. Apparatus for detecting traversal of a side slope by a vehicle, the apparatus including an accelerometer arranged to produce an acceleration signal indicative of the lateral acceleration of the vehicle, speed measuring means arranged to produce a speed dependent signal dependent on the speed of the vehicle and control means, characterized in that the control means is arranged to monitor the acceleration signal and the speed dependent signal and to produce a side slope detection signal based on a comparison between them.
2. Apparatus according to claim 1 wherein the speed dependent signal is arranged to vary with the square of the vehicle speed.
3. Apparatus according to claim 1 or claim 2 further comprising comparator means arranged to produce a difference signal indicative of the difference between the speed dependent signal and the acceleration signal.
4. Apparatus according to claim 3 further comprising integrating means arranged to integrate the difference signal over time and to produce the side slope detection signal if the integral reaches a predetermined value.

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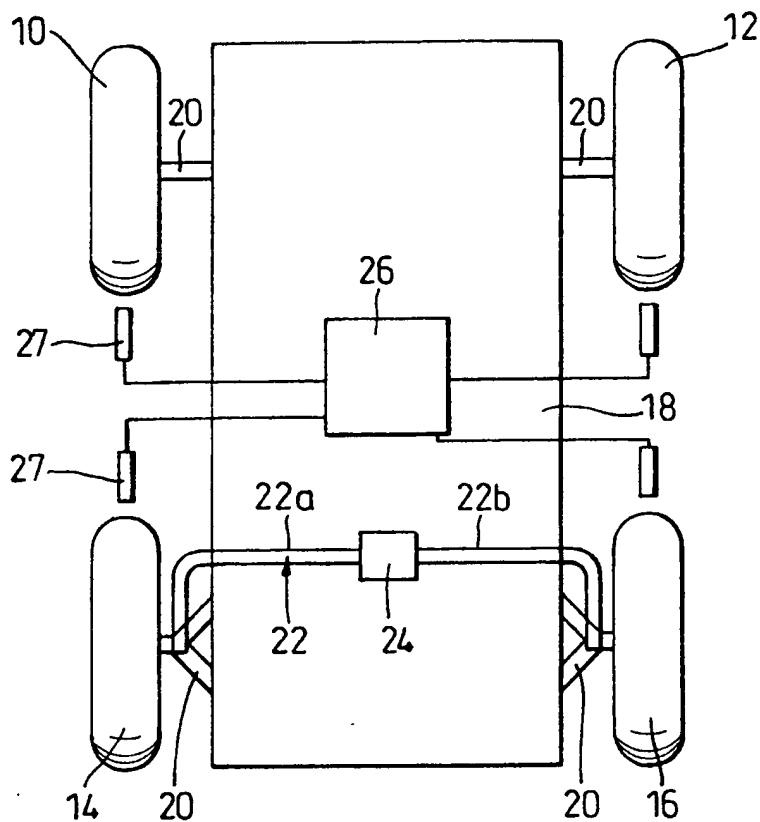
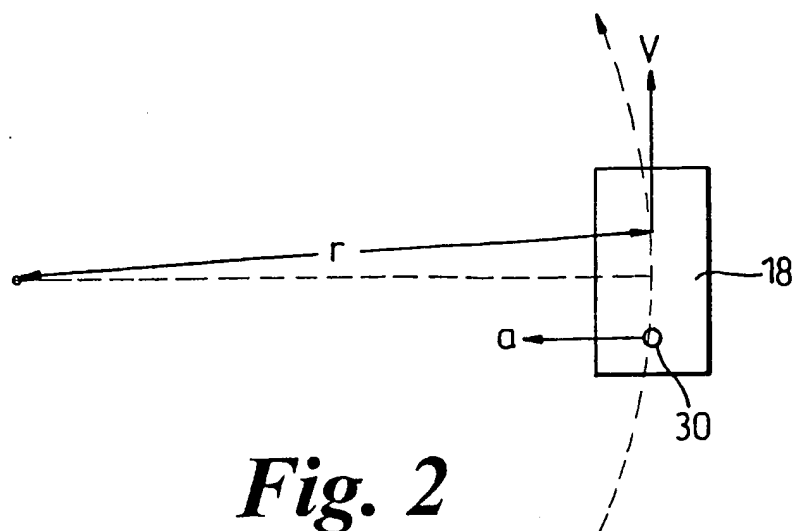
5. A vehicle suspension system including roll control means for controlling roll of the vehicle, the roll control means having active and passive modes, characterized in that the system further comprises side slope detection means arranged to detect when the vehicle is traversing a side slope, and the roll control means is arranged to respond to detection of a side slope by operating in the passive mode.
6. A system according to claim 5 wherein the roll control means includes a hydraulic circuit including a pump which is arranged to provide hydraulic pressure for use in active roll control when the roll control means is in the active state but which can be turned off or operate at reduced power when the roll control means is in the passive state.
7. A system according to claim 5 or claim 6 wherein the roll control means includes a roll bar having two parts which can be moved relative to each other to control roll when the roll control means is in the active state and which can be locked together when the roll control system is in the passive state.
8. A system according to any one of claims 5 to 7 wherein the side slope detection means comprises apparatus according to any one of claims 1 to 4.

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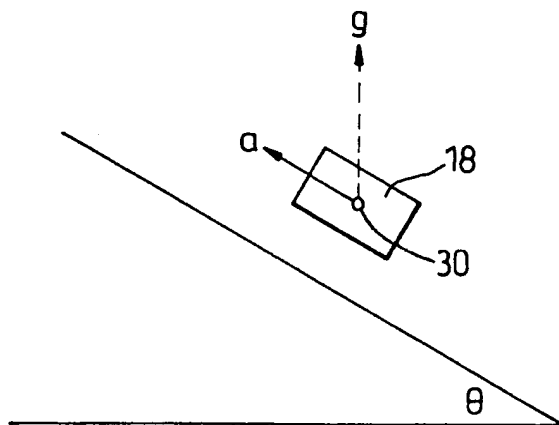
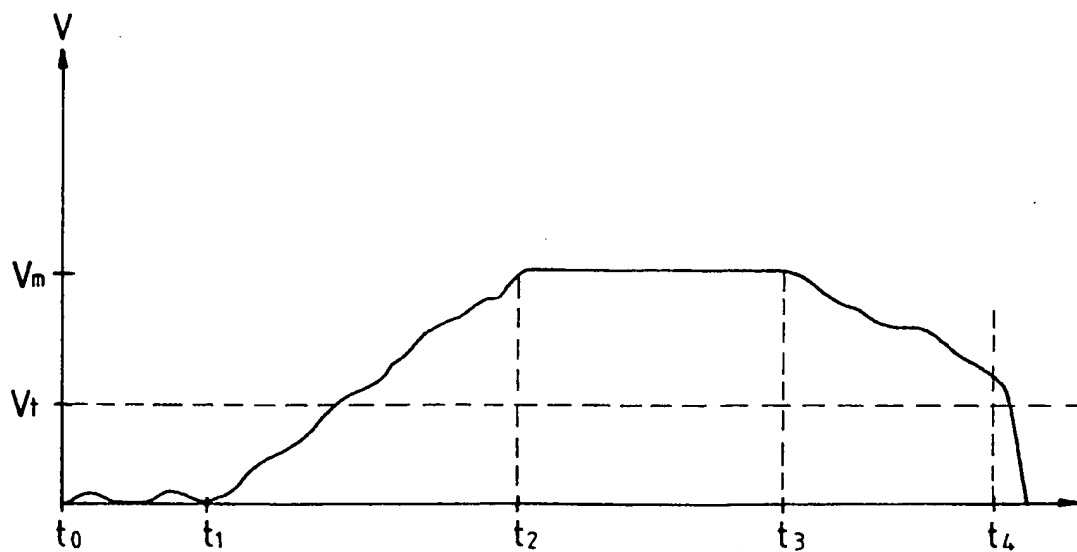
9. A system according to claim 8 wherein the roll control means is arranged to switch from the passive mode to the active mode if the vehicle speed exceeds a predetermined threshold speed while the roll control means is in the passive mode.
10. A system according to claim 9 further comprising means for producing a high speed signal when the vehicle speed exceeds said threshold speed, and wherein the high speed signal is input to the comparator means which is arranged to respond by altering the difference signal as if a speed signal indicative of a high vehicle speed or a lateral acceleration signal indicative of low lateral acceleration had been received.
11. Apparatus for detecting traversal of a side slope by a vehicle substantially as hereinbefore described with reference to the accompanying drawings.
12. A vehicle suspension system substantially as hereinbefore described with reference to the accompanying drawings.



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*Fig. 1**Fig. 2*

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*Fig. 3**Fig. 5*

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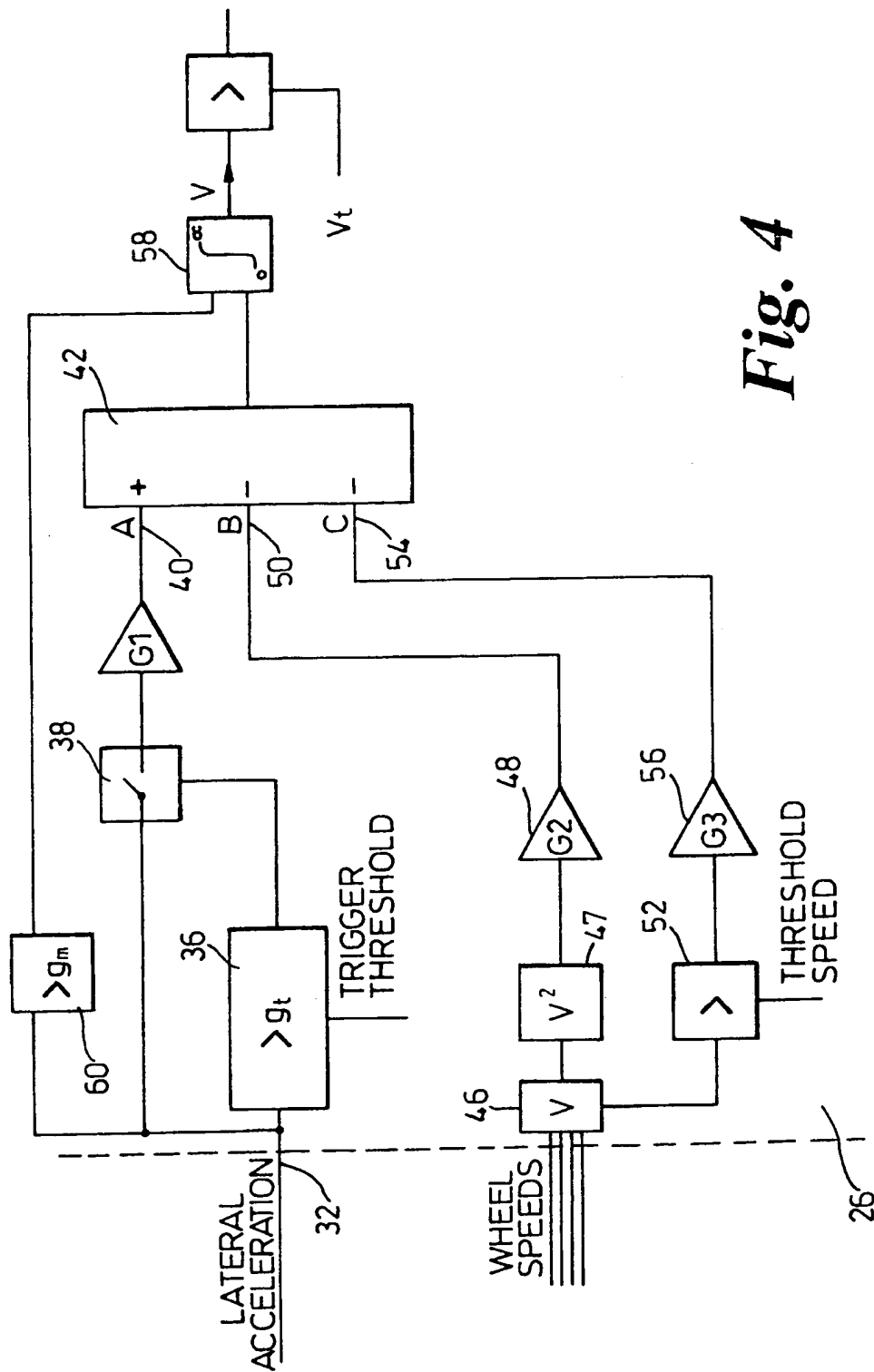


Fig. 4

# INTERNATIONAL SEARCH REPORT

International Application No.

PC/GB 99/01714

**A. CLASSIFICATION OF SUBJECT MATTER**  
IPC 6 B60G21/055 B60G17/015

According to International Patent Classification (IPC) or to both national classification and IPC

**B. FIELDS SEARCHED**

Minimum documentation searched (classification system followed by classification symbols)

IPC 6 B60G

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

**C. DOCUMENTS CONSIDERED TO BE RELEVANT**

Category	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	EP 0 689 116 A (GENERAL MOTORS) 27 December 1995 (1995-12-27) page 5, line 5 - line 11; figures 4,5 ---	1
X	DE 196 07 050 A (ITT AUTOMOTIVE EUROPE) 7 August 1997 (1997-08-07) claim 1 ---	1
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A	DE 40 19 732 A (FUJI JUKOGYO) 10 January 1991 (1991-01-10) abstract ---	5,6
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Further documents are listed in the continuation of box C.



Patent family members are listed in annex.

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# INTERNATIONAL SEARCH REPORT

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## C.(Continuation) DOCUMENTS CONSIDERED TO BE RELEVANT

Category	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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